Proposal for the Annex C of the new UN GTR on Laboratory Measurement of Brake Emissions for Light-Duty Vehicles

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I. Statement of technical rationale and justification

II. Text of the Annex C

1. Purpose

This document provides the procedure to determine vehicle-specific friction braking share coefficients for use with the Global Technical Regulation on the measurement of brake wear particulate matter and particle number emissions from brakes used on Light-Duty vehicles. The method described in this annex may be used as an alternative to the friction braking share coefficients of the GTR, given in paragraph 5.2 table 5.1.

2. Scope and application

The GTR on Brake Particle Emissions is applied to all types of light duty vehicles. The method described in this Annex C shall be applied to all vehicle types with non-friction braking capabilities. It is meant as enhancement for Table 5.1., and describes the alternative, vehicle specific method for establishing of the Friction Braking Share Coefficients for vehicle types NOVC-HEV Cat.1, NOVC-HEV Cat.2, OVC-HEV and PEV.

This annex C describes the method setup and procedures for running WLTP-Brake Cycle tests on a chassis dynamometer and how to determine the vehicle-specific friction braking coefficient. It states a procedure and acceptance criteria for the use of alternative methods. Furthermore, it provides a procedure how to transform the vehicle specific friction share measured of the vehicle running on the WLTP-Brake-Trip10, or the WLTC (Exhaust) to vehicle-specific friction braking share coefficient of the identical vehicle running on WLTP-Brake Cycle.

[3. Definitions

3.1. Vehicle and Brake Dynamometer settings

3.1.1."Road Loads" means the total force or power required to move the vehicle on a level and smooth surface at a specified speed and mass. Road loads take account of the frictional losses of the drivetrain. In this UN GTR, a reduction of the brake nominal inertia by a fixed percentage of 13 per cent is considered to account for road loads in full-friction braking emissions testing.

3.1.2. <u>"Friction Brake Share Coefficient</u>" is the ratio of total energy absorbed by the full friction brake system during a drive cycle, to the total kinetic energy variation of the vehicle during braking events (less road load) over the same drive cycle.

3.1.3. <u>"Not off-vehicle charging hybrid electric vehicle" (NOVC-HEV) means a hybrid electric vehicle that cannot be charged from an external source. In this UN GTR, NOVC-HEV are categorised to "NOVC-HEV Category 1" and "NOVC-HEV Category 2" based on their traction REESS nominal voltage.</u>

3.1.4 "Not off-vehicle charging hybrid electric vehicle – Category 1" (NOVC-HEV Cat. 1) means a hybrid electric vehicle that features a traction REESS with a nominal voltage higher than 20V and lower than or equal to 60V that cannot be charged from an external source.

3.1.5 "Not off-vehicle charging hybrid electric vehicle – Category 2" (NOVC-HEV Cat. 2) means a hybrid electric vehicle that features a traction REESS with a nominal voltage higher than 60V that cannot be charged from an external source.

3.1.6 "Off-vehicle charging hybrid electric vehicle" (OVC-HEV) means a hybrid electric vehicle that can be charged from an external source. 3.7.10. "Pure electric vehicle" (PEV) means a vehicle equipped with a powertrain containing exclusively electric machines as propulsion energy converters and exclusively rechargeable electric energy storage systems as propulsion energy storage systems.

3.2. Test system

3.2.1. "Torque measurement sensor" means an electromechanical device that converts the torsional strain on the brake assembly into the equivalent output. The equivalent torque derives from the angular deceleration rate and the effective brake inertia. Alternatively, a device can be used to measure the force at the brake calliper and calculate the according torque by use of the correct geometric parameters.

3.2.2. Pressure sensor" for this annex means an electromechanical device that is connected to the brake fluid path close to the brake system and provides a signal that is equivalent to the brake pressure at the corresponding brake corner.

3.2.3. "Torque <u>to Pressure</u> ratio" (C_p) is a constant <u>value</u> that converts the brake pressure (Pa) into braking torque (Nm) of a friction brake.

3.2.4 "Torque to Electric Power ratio" (C_e) is a transfer function that converts measured electric power into braking torque (Nm) of an electro-mechanical friction brake.

3.2.5. <u>"Tyre dynamic rolling radius"</u> means the tyre radius that equates to the revolutions per kilometre (or revolutions per mile) published by the tyre manufacturer for the specific tyre size (mm).

3.2.6. "Vehicle speed" is the vehicle speed as given by the drive curve of the chassis dynamometer

3.2.7 "Set vehicle speed" corresponds to the setpoint of the vehicle speed at a certain time of the test

3.2.8. <u>"Chassis Dynamometer"</u> means a technical system that imposes and controls a drive schedule on a complete vehicle, while recording relevant data.

4. Abbreviations and Symbols

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Symbol/Abbreviation	Definition	Unit	Paragraph
C_p	Torque to Pressure ratio converting measured brake pressure (Pa) into braking torque (Nm)	Nm/Pa	5.2, 5.3, 5.3.3
С	Friction braking share coefficient calculated from WLTP-Brake cycle	-	5.1, 8.1, 8.2
Ctrip10	Friction braking share coefficient can be calculated from WLTP-Brake-Trip-10 cycle	-	8.1
Cexhaust	Friction braking share coefficient calculated on WLTC (Exhaust) cycle	-	8.2
Calt	Friction braking share coefficient calculated from WLTP-Brake cycle through an alternative method	-	7.3
Ce	torque to electric power ratio	Nm/W	5.2, 5.3
EMB	Electro-Mechanical Brake	-	6.1.2
GTR	Global Technical Regulation	-	5, 5.2, 5.3, 6.2, 6.3, 6.4
kbr,trip10	Correlation factor between WLTP-brake cycle and WLTP-Brake-Trip-10 cycle	-	8.1
k _{br,ex}	correlation factor between WLTP-brake cycle and WLTC (Exhaust) cycle	-	8.2
MRO	Mass in Running Order	kg	6.2.1
NOVC-HEV	Not Off-Vehicle Charging Hybrid Electric Vehilce	-	6.4, 7.1
OVC-HEV	Off-Vehicle Charging Hybrid Electric Vehicle	-	6.4, 7.1
P _{brake}	Measured electric power of electromechanical full-friction brakes	W	5.2
p_{brake}	Measured braking pressure	bar	5.2
PEV	Pure Electric Vehicle	-	6.4, 7.1
ľR	Tyre Dynamic rolling radius	m	5.2
t	duration of a braking event	s	5.2
Wbrake	Friction work, which is the sum of the energy dissipated in the full friction brake system during all braking events in the relevant schedule	J	5.1, 5.2, 7.3
Wtotal,bc	Sum of the kinetic energy variation of the vehicle during all braking events. For the WLTP-Brake Cycle W _{total,bc} is calculated by multiplying the vehicle test mass M _{veh} with 15983 J/kg	J	5.1
W _{ref,bc}	$\begin{array}{llllllllllllllllllllllllllllllllllll$	J	5.1

Symbol/Abbreviation	Definition	Unit	Paragraph
Wtotal, Trip10	Sum of the kinetic energy <u>variation</u> of the vehicle during all braking events of WLTP-Brake-Trip10. It is calculated by multiplying the vehicle test mass M _{veh} by 5555 J/kg	J	5.1
Wref, Trip10	Sum of kinetic energy <u>variation</u> during all braking events of WLTP-Brake-Trip10 (W _{total, Trip10}) reduced by road load contribution share of arbitrary 13%	J	5.1
Wref, Exhaust	Sum of kinetic energy <u>variation</u> during all braking events of the WLTC (Exhaust). It is calculated by multiplying the vehicle test mass M _{veh} by 3578 J/kg.	1	5.1
WLTP	Worldwide harmonized Light vehicle Test Procedure	-	5
v	Measured vehicle speed	m/s	5.2, 5.3

5. <u>Reference Method Overview and Calculation Method</u>

The vehicle individual friction share coefficient shall be determined on a fully GTR15 compliant chassis dynamometer applying the WLTP Brake test cycle. All brakes shall be equipped with external sensors to determine the brake torque at each of the wheels.

5.1 Calculation Method of Friction <u>Braking</u> Share <u>coefficient</u> c

In the main GTR the brake particle emission factors are determined on a brake dynamometer, which is set to the respective road load considering reducing the inertia by 13% of the vehicle mass. In order to be consistent, the friction share coefficient is calculated as ratio of "stopping energy dissipated by the friction brakes" divided by "total stopping energy less road loads (defined as 13% in the GTR)".

Friction braking share coefficient

$$c = \frac{W_{brake}}{W_{ref}}$$

(Eq. 1)

Where:

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с	is the friction braking share coefficient.
W _{brake}	is the sum of energy dissipated at the full-friction brake during all braking events
	in J;
W _{ref, bc}	is sum of kinetic energy variation during all braking events of the WLTP-Brake
	<u>cycle ($W_{total, bc}$</u>) reduced by arbitrary <u>13% to consider</u> road load_losses, $W_{ref, bc}$ =
	$W_{\text{total, bc}} * 0.87 \text{ in J.}$
Wref, Trip10	is sum of kinetic energy variation during all braking events of WLTP-Brake-
	Trip10 (W _{total, Trip10}) reduced by arbitrary 13% to consider road load losses, W _{ref} ,
	$Trip10 = W_{total, Trip} * 0.87 \text{ in J.}$
Wref, Exhaust	is sum of kinetic energy variation during all braking events of the WLTC
	(Exhaust) (W _{total, Exhaust}) in J.

W _{total, bc}	is the sum of the kinetic energy <u>variation</u> of the vehicle during all braking
	events, of WLTP-Brake Cycle. The W _{total, bc} is calculated by multiplying the
	vehicle test mass M _{veh} by 15983 J/kg.
Wtotal, Trip10	is the sum of the kinetic energy variation of the vehicle during all braking
	events. of WLTP-Brake-Trip 10. The Wtotal, Trip10 is calculated by multiplying
	the vehicle_test mass M _{veh} by 5555 J/kg.

In case of WLTC (Exhaust) $W_{ref, exhaust}$ is_calculated by multiplying the vehicle test mass M_{veh} by 3578 J/kg.

The friction share coefficient is determined for the vehicle running on the WLTP-Brake Cycle. Alternatives and acceptance criteria are described in paragraph 8 of this annex.

5.2. Method to determine the friction work Wbrake

The energy dissipated at hydraulic full-friction brakes_is calculated from the measured brake pressure (p_{brake}) multiplied by the <u>torque to pressure</u> ratio (C_p) at the respective front and rear brakes (representing front and rear axle) during the brake applications of the driving cycle.

[Note: In the following the formula are written not yet specified for front and rear brakes. This needs to be separated into front axle and rear axles].

$$W_{\text{brake}} = C_p \sum_{i=1}^{N} \int_{0}^{t_{i-th}} p_{\text{brake}} \cdot 10^5 \cdot \frac{v}{r_R} dt$$

(Eq. 2)

Where:	
W _{brake}	is the sum of energy dissipated at the full-friction brake during all braking events
	in J;
Cp	is the Torque to pressure ratio in Nm/Pa;
p _{brake}	is the brake pressure measured in bar;
v	is the vehicle speed as given by the drive curve in m/s;
r <u>R</u>	is the tire dynamic rolling radius in m;
t	is the duration of a braking event in s.

The energy dissipated at electromechanical full-friction brakes is calculated from the measured electric power (P_{brake}) multiplied by the <u>torque to electric power</u> ratio (Ce) at the respective front and rear brakes (representing front and rear axle) during the brake applications of the driving cycle.

[Note: the formula to calculate Wbrake for electromechanical brakes has to be added]

5.3 Determination of C_p and C_e value

The determination of the C_p value, or C_e value for a specific brake system is based on a test run applying the WLTP-Brake Cycle. The test must be run on a brake dynamometer fully complying to this GTR.

5.3.1 Calibration of Brake Dynamometer

The brake dynamometer and all testing equipment should be setup and operated according to this GTR.

5.3.2. Operation

- 1. Install the brake according to the measurement procedure of this GTR.
- 2. Run WLTP-Brake test according to this GTR
- 3. Depending on brake type, record:
 - a. brake torque and brake pressure for hydraulic or electro-hydraulic brake, or
 - b. brake torque and electric power (current, voltage and additional required signals) for an electromechanical brake.
 Note: Usage of "Slow sampling rate" channels for the C_p evaluation as per 3.1.29. is recommended.

5.3.3. C_p calculation

Cp value describes the relationship between brake pressure and brake torque:

$$Cp = \frac{Brake Torque}{Brake Pr essure}$$
 (Eq. 3)

For a given friction material, C_p may depend on vehicle speed, applied brake pressure, brake rotor and pad temperature. It may change from <u>brake</u> snub to <u>brake</u> snub during the execution of the test. To reduce the influence of C_p variability on brake energy calculation in the test cycle, "energy weighted Cp value" according to the formula below is used:

If the applied brake pressure is smaller or equal to 0.5 bar, the data point will not be considered for C_p calculation for decelerations events (a(t) <0). If the applied brake pressure is larger than 0.5 bar, C_p is calculated as follows for deceleration events (a(t)<0):

$$Cp_{avg} = \frac{\int_{t=0}^{End} v(t) \times a(t) \times \frac{Tq(t)}{p(t)} dt}{\int_{t=0}^{End} v(t) \times a(t) dt} \qquad (Eq. 4)$$

In case of running WLTP-Brake-Trip 10 on the Chassis Dynamometer for establishing of the friction share (see paragraph 8.1.) the Cp or Ce evaluation as described above should be carried out for data of WLTP-Brake-Trip 10.

5.3.4. Ce calculation

Torque to Electric Power ratio" (Ce) is a transfer function that converts measured electric power into braking torque (Nm) of an electro-mechanical friction brake.

<u>(...)</u>

6. Testing Setup and Specifications

6.1. Vehicle preparation

6.1.1 Pressure transducers and sensors and its calibration

For this method, an external pressure sensor shall be mounted to the brake fluid path for each brake corner of the vehicle. Preferably, it shall be mounted to the venting screw of the respective brake corner. If this is not possible due to space limitations or other issues, alternative mounting locations are allowed, however they should be located as close as possible to the according brake corner. The measured pressure over time shall be converted to brake energies and to c-factors according to the equation 1 and 2 described in chapter 5.1. and 5.2.

A range of 0 - [20000 kPa] is recommended, while the maximum range of the sensor shall be chosen according to the existing maximum pressure conditions during the tests.

Sensor calibrations shall meet the following specifications:

 uncertainty of measurement for pressure measurement- [0.5%] of reading or [0.1%] of full scale whichever is larger (entire uncertainty budget)

To avoid zero drift, the pressure sensors shall be adjusted to zero before the test with no brake pressure applied to the system.

After the test, the pressure sensors shall be checked for zero drift. A max. zero-drift of [+/- XXX Pa] is acceptable.

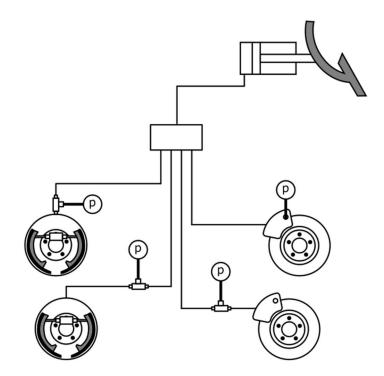


Figure 6.1. Schematic mounting of brake pressure sensors (P) at brake pipes of the tested vehicle

6.1.2. Sensors for force measurements on electro-mechanical brake systems (EMB) and its calibration

If the brake systems of the vehicle operate without brake fluid, it is required to derive the braking energies from different sources. For EMB it is required to measure the electric current and voltage to calculate the brake power and convert it to brake energy according to equations Eq. [2a] in chapter 5.2.

Measurement ranges of [0 - ... V] and [0 - ... A] are recommended, and the sensors shall meet the following specifications:

[more details to be added:]

6.2. Data recording

6.2.1. Chassis Dynamometer Data

During testing, dedicated data recording systems have to be used to log the raw data from the chassis dynamometer as well as from the vehicle and its instrumented components.

The data recording recommendations are referred to GTR 15 (Annex B5, chapter 2 Chassis Dynamometer).

Additionally to the data requested in GTR15, foundation brake related parameters have to be recorded. This includes at least the brake pressure in bar. The measurement has to be done with a frequency of not less than 10 Hz.

Regarding the vehicle masses, different specifications for the test mass have to be fulfilled:

- According to 3.2.25 of GTR15 for WLTC(Exhaust)
- According to 3.1.9 of this GTR for WLTP-Brake Cycle
- According to 3.1.9 of this GTR for WLTP-Brake-Trip10

Moreover, the following input parameters must be documented:

- <u>Tire dynamic rolling radius </u><u>*r*_R</u>
- <u>Torque to Pressure ratio</u> C_p values

The data recording of dynanometer and vehicle has to ensure synchronized data, meaning that the signals have to refer to the same time trace. It is recommended to record the signals time-aligned on a single file. Alternatively, the vehicle speed signal shall be recorded together with the brake information and be used for time alignment, if the data is recorded on different systems.

Recorded data has to be provided in a common and open access data format.

6.3 Chassis Dynanometer Settings

The tests are setup and performed as described and specified in the currently valid version of UN GTR No. 15 as available at (https://wiki.unece.org/display/trans/Latest+GTR+15). For brake energy determination the 4-phase WLTC Class 3 cycle shall be used. No deviations, except the definitions in this document, are valid.

6.3.1. Consideration of Road Loads

For both test cycles, WLTC(Exhaust) and WLTP_Brake Cycle, the road load simulation is considered to be fully compliant to GTR No. 15. <u>Soaking, pre-conditioning and Road according to GTR.</u> This means, that the road load coefficients (f0, f1, f2) of the road load equation are taken into account for the test execution.

6.4._____Test Sequence WLTP Brake

The test vehicle shall be run-in in accordance with the requirements in paragraphs 2 to 2.3. of Annex 8 to UN GTR No. 15

Generally, the test is carried out by applying the sequence of preconditioning, soaking and (for OVC-HEV and PEV) recharging, followed by the performance test to derive the friction braking share (See Figure 6.4.). The requirements for those shall be as set out in Annex 8 of UN GTR No. 15, and particularly the following sections corresponding to the vehicle type being tested:

- NOVC-HEV (Cat.1 and Cat.2) paragraphs 3.3.1.1.to 3.3.3.1.
- OVC-HEV paragraphs 2.2. to 2.2.3.2. of Appendix 4 and paragraphs 3.2.4.2.1. and 3.2.4.2.2.
- PEV paragraphs 3.4.2. to 3.4.4.1.

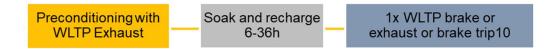


Figure 6.4. Principle Structure of chassis dyno testing according to this section

Notwithstanding the above requirements, the applicable test cycle during the performance test shall be the WLTP-Brake Cycle as described in Annex A. For OVC-HEV and PEV the cycle shall be driven only once.

In line with the provisions in paragraph 8 of this Annex the applicable test cycle above may be replaced by either:

- the WLTP Trip #10 of the WLTP-Brake Cycle, or;
- the WLTP(Exhaust) cycle, being the 4-phase test cycle Class 3b as prescribed in paragraphs 3.3.2. to 3.3.2.5. of Annex 1 to UN GTR No. 15.

Speed trace tolerances shall be the ones described in (a) and (b) of paragraph 9.4.1. of this GTR. The number of speed violations shall not exceed a number corresponding to 3% of the applicable test cycle (as defined above) duration.

In case the vehicle cannot follow the speed trace of any of the cycles above, the friction brake share in Table 5.1. of this GTR shall be used by default.

7. Equivalence of Methods

At the option of the manufacturer and if the equivalence criterion described in paragraph 7.3 is fulfilled, an alternative method may be used for the determination of the individual friction share coefficient instead of the method described in paragraph 5.

7.1. Selection of Vehicle and Electrification Concept for Proof of Equivalence

The manufacturer shall demonstrate the equivalence of the alternative method to the reference method for each powertrain category (i.e. the familiy head for NOVC-HEV Cat.1, NOVC-HEV Cat2, OVC-HEV, and PEV).

7.2 Testing of the Alternative Method.

In order to prove equivalence, the vehicle shall be equipped with brake pressure transducers and sensors <u>according to</u> paragraph 6.1 and shall be subjected to the WLTP-Brake Cycle according to the test sequence <u>defined in</u> paragraph 6.4.

7.3. Equivalence Criterion

The alternative method shall be deemed to be equivalent to the reference method if <u>one of</u> the following conditions is fulfilled:

 $\left|\frac{c_{alt} - c}{c}\right| \le [10\%] \quad (Eq.5)$ $|c_{alt} - c| \le [x\%] \quad (Eq.6)$

Where:

 c_{alt} is the friction braking share coefficient measured through the alternative method and calculated according to Eq. 1

<u>c</u> is the friction braking share coefficient measured through the reference method and calculated according to Eq. 1

8. Equivalence of Test Cycle

As an alternative to derive the friction braking share coefficient from WLTP-brake cycle, the manufacturer may choose to calculate it from WLTP-Brake-Trip-10 cycle or from WLTC (Exhaust) cycle.

8.1. WLTP-Brake- Trip 10

8.1.1. Calculation of Friction share from WLTP-Brake-Trip-10 cycle

The friction braking share coefficient can be calculated from WLTP-Brake-Trip-10 cycle as follows:

$$c = c_{trip10} \cdot k_{br,trip10} \qquad (Eq. 7)$$

where:

 c_{trip10} is the friction braking share coefficient calculated on WLTP-Brake_Trip-10 cycle according to Eq.1 where the friction work W_{brake} is calculated either according to Eq. 2 or with an alternative method as described in paragraph 7.

k_{br,trip10} is a [constant] correlation factor between WLTP-Brake cycle and WLTP-Brake-Trip-10 cycle.

8.2. WLTC (Exhaust)

8.2.1. Calculation of Friction Share from WLTC (Exhaust) <u>cycle</u> The friction braking share coefficient can be calculated from WLTC (Exhaust) <u>cycle</u> as follows:

$$c = c_{exhaust} \cdot k_{br,ex} \qquad (Eq. 8)$$

where:

 $c_{exhaust}$ is the friction braking share coefficient calculated on WLTC (Exhaust) <u>cycle</u> according to Eq.1 where the friction work W_{brake} is calculated either according to Eq. 2 or with an alternative method as described in paragraph 7.

kbr,ex is a [constant] correlation factor between WLTP-brake cycle and WLTC (Exhaust) cycle.

If $c_{exhaust}$ is lower than [x%], Eq. 8 shall not be used and the friction braking share coefficient shall be calculated either from WLTP-Brake cycle or from WLTP-Brake-Trip-10 cycle according to paragraph 8.1.2.

8.3. Offset of the friction braking share coefficient ("declaration")

The friction braking share coefficient calculated according to paragraphs 8.1. and 8.2. may be increased by the manufacturer to cover the statistical <u>and procedural</u> uncertainties related to the use of the correlation factors in Eq. 7 or in Eq. 8,

9. Test Output

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